

**Table 3 Suction, Discharge, and Liquid Line Capacities in Tons for Refrigerant 22 (Single- or High-Stage Applications)**

Line Size Type L Copper, OD		Suction Lines ( $\Delta t = 2^\circ\text{F}$ )				Discharge Lines ( $\Delta t = 1^\circ\text{F}$ , $\Delta p = 3.05 \text{ psi}$ )		Line Size Type L Copper, OD	Liquid Lines See notes a and b			
		-40	-20	0	20	40	Saturated Suction Temperatures, $^\circ\text{F}$		Type L Copper, OD	Vel. = 100 fpm	$\Delta t = 1^\circ\text{F}$ $\Delta p = 3.05$	
		Corresponding $\Delta p$ , psi/100 ft				-40	40					
1/2	—	—	—	—	0.40	0.6	0.75	0.85	1/2	2.5	3.6	
5/8	—	0.32	0.51	0.76	1.1	1.4	1.6	5/8	3.7	6.7		
7/8	0.52	0.86	1.3	2.0	2.9	3.7	4.2	7/8	7.6	18.2		
1 1/8	1.1	1.7	2.7	4.0	5.8	7.5	8.5	1 1/8	13.2	37.0		
1 3/8	1.9	3.1	4.7	7.0	10.1	13.1	14.8	1 3/8	20.2	64.7		
1 5/8	3.0	4.8	7.4	11.1	16.0	20.7	23.4	1 5/8	28.5	102.5		
2 1/8	4.7	7.6	11.6	17.1	25.1	32.1	37.4	2 1/8	49.6	213.0		
2 3/8	7.0	11.0	17.0	25.0	36.0	45.4	52.4	2 3/8	76.5	376.9		
3 1/8	11.5	18.4	28.0	44.0	65.0	82.9	100.2	3 1/8	109.2	601.5		
3 3/8	16.0	25.3	38.4	58.6	87.8	111.4	136.2	3 3/8	147.8	895.7		
4 1/8	24.8	39.6	59.2	88.3	131.3	168.4	211.1	4 1/8	232.1	1,263.2		
Steel								Steel				
IPS	SC&H					IPS	SC&H					
1/2	40	—	0.58	0.58	0.83	1.2	1.5	1.7	1/2	800	3.8	5.7
3/4	40	0.50	0.8	1.2	1.8	2.5	3.3	3.7	3/4	800	6.9	12.8
1	40	0.95	1.5	2.3	3.4	4.8	6.1	6.9	1	800	11.5	25.2
1 1/4	40	2.0	3.2	4.8	7.0	9.9	12.6	14.3	1 1/4	800	20.6	54.1
1 1/2	40	3.0	4.7	7.2	10.5	14.8	19.0	21.5	1 1/2	800	28.3	82.6
2	40	5.7	9.1	13.9	20.2	28.5	36.8	41.4	2	400	53.8	192.0
2 1/2	40	9.2	14.6	22.1	32.2	45.4	58.1	65.9	2 1/2	400	78.7	305.8
3	40	14.2	23.7	36.0	54.8	80.1	102.8	116.4	3	400	118.5	540.3
4	40	23.1	37.5	56.8	85.9	124.3	160.5	187.3	4	400	204.2	1,001.2

- Notes:  
 1. Table capacities are in tons of refrigeration.  
 $\Delta p$  = pressure drop from line friction, psi per 100 ft of equivalent line length  
 $\Delta t$  = corresponding change in saturation temperature,  $^\circ\text{F}$  per 100 ft  
 2. Line capacity for other saturation temperatures  $\Delta t$  and equivalent lengths  $L_e$   

$$\text{Line capacity} = \text{Table capacity} \left( \frac{\text{Table } L_e}{\text{Actual } L_e} + \frac{\text{Actual } \Delta t}{\text{Table } \Delta t} \right)^{1.8}$$
  
 3. Saturation temperature  $\Delta t$  for other capacities and equivalent lengths  $L_e$   

$$\Delta t = \text{Table } \Delta t \left( \frac{\text{Actual } L_e}{\text{Table } L_e} \right) \left( \frac{\text{Actual capacity}}{\text{Table capacity}} \right)^{1.8}$$

\*Using values is recommended where any gas generated in receiver must return up condenser line to condenser without reversing condenser flow. Water-cooled condensers, where receiver ambient temperature may be higher than refrigerant condensing temperature, fall into this category.

4. Values based on 105 $^\circ\text{F}$  condensing temperature. Multiply table capacities by the following factors for other condensing temperatures.

Condensing Temperature, $^\circ\text{F}$	Suction Line	Discharge Line
80	1.11	0.79
90	1.07	0.88
100	1.03	0.95
110	0.97	1.04
120	0.90	1.10
130	0.84	1.18
140	0.80	1.26

\*Line pressure drop  $\Delta p$  is conservative, if subcooling is substantial or line is short, a smaller size line may be used. Applications with very little subcooling on very long lines may require a larger line.

**Table 4 Suction, Discharge, and Liquid Line Capacities in Tons for Refrigerant 22 (Intermediate- or Low-Stage Duty)**

Line Size Type L Copper, OD		Suction Lines ( $\Delta t = 2^\circ\text{F}$ ) <sup>a</sup>					Discharge Lines ( $\Delta t = 2^\circ\text{F}$ ) <sup>a</sup>	Liquid Lines	
		Saturated Suction Temperature, $^\circ\text{F}$							
		-90	-80	-70	-60	-50	-40	-30	
5/8	—	—	—	—	—	—	—	—	0.7
7/8	—	0.18	0.25	0.34	0.46	0.61	0.79	1.0	1.9
1 1/8	—	0.34	0.51	0.70	0.94	1.2	1.6	2.1	3.8
1 3/8	—	0.6	0.9	1.2	1.6	2.2	2.8	3.6	6.6
1 5/8	—	1.0	1.4	1.9	2.6	3.4	4.3	5.7	10.5
2 1/8	—	2.1	3.0	4.1	5.5	7.2	9.3	11.9	21.7
2 3/8	—	3.8	5.3	7.2	9.7	12.7	16.5	21.1	38.4
3 1/8	—	6.1	8.5	11.6	15.5	20.4	26.4	33.8	61.4
3 3/8	—	9.1	12.7	17.3	23.1	30.4	39.4	50.2	91.2
4 1/8	—	12.9	18.0	24.3	32.7	43.0	55.6	70.9	128.6
5 1/8	—	20.2	27.3	36.9	49.9	65.1	84.8	108.9	209.5
6 1/8	—	31.5	42.1	56.8	74.6	97.4	126.5	162.2	349.4

- Notes:  
 1. Table capacities are in tons of refrigeration.  
 $\Delta p$  = pressure drop from line friction, psi per 100 ft of equivalent line length  
 $\Delta t$  = corresponding change in saturation temperature,  $^\circ\text{F}$  per 100 ft  
 2. Line capacity for other saturation temperatures  $\Delta t$  and equivalent lengths  $L_e$   

$$\text{Line capacity} = \text{Table capacity} \left( \frac{\text{Table } L_e}{\text{Actual } L_e} + \frac{\text{Actual } \Delta t}{\text{Table } \Delta t} \right)^{1.8}$$
  
 3. Saturation temperature  $\Delta t$  for other capacities and equivalent lengths  $L_e$   

$$\Delta t = \text{Table } \Delta t \left( \frac{\text{Actual } L_e}{\text{Table } L_e} \right) \left( \frac{\text{Actual capacity}}{\text{Table capacity}} \right)^{1.8}$$
  
 4. Refer to refrigerant thermodynamic property tables (Chapter 20 of the 2013 ASHRAE Handbook—Fundamentals) for pressure drop corresponding to  $\Delta t$ .

<sup>a</sup>See section on Pressure Drop Considerations.

5. Values based on 0 $^\circ\text{F}$  condensing temperature. Multiply table capacities by the following factors for other condensing temperatures. Flow rates for discharge lines are based on -50 $^\circ\text{F}$  evaporating temperature.

Condensing Temperature, $^\circ\text{F}$	Suction Line	Discharge Line
-30	1.09	0.58
-20	1.06	0.71
-10	1.03	0.83
0	1.00	1.00
10	0.97	1.20
20	0.94	1.43
30	0.90	1.80

# Ashrae Chapter 26

**Robert R. Redfield**



## Ashrae Chapter 26:

**Ventilation and Infiltration** American Society of Heating, Refrigerating and Air-Conditioning Engineers, 2001

*Handbook of Climate Change Mitigation and Adaptation* Maximilian Lackner, Baharak Sajjadi, Wei-Yin Chen, 2025-09-26

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*Occurrence of Mold in a Two-story Wood-frame House Operated at Design Indoor Humidity Levels*, 2009 Mold growth was observed in limited areas in a two story contemporary wood frame house during the seventh heating season of operation at or near design indoor humidity levels The house was in a cold climate Madison Wisconsin Humidity levels were estimated as being exceeded in one of 10 homes at this location Moderate amounts of window condensation were observed during each of the seven heating seasons Areas of observed mold growth were limited to the lower extremities of windows near the lower edge of glass panes and the edges of lower panels of a wood panel entry door that was not equipped with a storm door Mold growth was also present in sprayed on cellulose insulation in close proximity to the rim joist board in the basement although this mold was not visible unless the insulation was disturbed and was probably present during previous heating seasons Mold from these locations was cultured isolated and identified morphologically or by DNA sequence Seven genera of common ascomycetes and deuteromycetes were detected all of which are commonly associated with indoor air Penicillium was the most common genus with at least six different species The greatest variety of genera five occurred in samples taken from interior wood millwork the edges of panels in the entry door

and from a wood window sash Only two genera were found in a sample taken from the interface between the rim joist board and cellulose insulation Total spore counts taken in April revealed that on the first and second stories mold spores were less prevalent than in outdoor air but that mold spores were more prevalent in the basement than in outdoor air *Masonry Walls for Energy Conservation* James E. Amrhein,1981 [Building Ventilation](#) Mat Santamouris,Peter Wouters,2006-06-01 Ensuring optimum ventilation performance is a vital part of building design Prepared by recognized experts from Europe and the US and published in association with the International Energy Agency s Air Infiltration and Ventilation Centre AIVC this authoritative work provides organized classified and evaluated information on advances in the key areas of building ventilation relevant to all building types Complexities in airflow behaviour climatic influences occupancy patterns and pollutant emission characteristics make selecting the most appropriate ventilation strategy especially difficult Recognizing such complexities the editors bring together expertise on each key issue From components to computer tools this book offers detailed coverage on design analysis and performance and is an important and comprehensive publication in this field Building Ventilation will be an invaluable reference for professionals in the building services industry architects researchers including postgraduate students studying building service engineering and HVAC and anyone with a role in energy efficient building design **Plant Engineer's Handbook of Formulas, Charts, and Tables** Donald W. Moffat,1982 *HVACR Principles and Applications* Nuggenhalli S. Nandagopal,2024-03-07 This book provides a clear and concise understanding of the principles and applications of HVACR using a rigorous yet easy to follow presentation The coverage is broad including relevant support areas such as fluid mechanics heat transfer thermodynamics psychrometrics with specific applications to HVACR design and calculations and main topics such as air conditioning processes cooling heating load calculations refrigeration cycles and HVACR equipment and systems The book integrates and illustrates the use of data and information from ASHRAE Handbooks and Standards in step by step calculations of cooling and heating loads and other aspects of HVACR Elucidation of the principles is further reinforced by examples and practice problems with detailed solutions Firmly grounded in the fundamentals the book maximizes readers capacity to take on new problems and challenges in the field of HVACR with confidence and conviction Providing a ready reference and review of essential principles and their applications in HVACR the book is ideal for HVACR practitioners undergraduate engineering students and those specializing in HVACR as well as for practicing engineers preparing for the engineering license exams FE and PE in USA and abroad The book uses both Inch Pound I P and S I systems of units to facilitate global readership and use **Solutions Manual for the Mechanical Engineering Reference Manual** Michael R. Lindeburg,1994 **Research in Building Physics and Building Engineering** Paul Fazio,Hua Ge,Jiwu Rao,Guyllaine Desmarais,2020-11-26 Buildings influence people They account for one third of energy consumption across the globe and represent an annual capital expenditure of 7% 10% of GNP in industrialized countries Their lifetime operation costs can exceed capital investment Building Engineering aims to make

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**Fundamentals of Building Energy Dynamics** Bruce D. Hunn,1996 Fundamentals of Building Energy Dynamics assesses how and why buildings use energy and how energy use and peak demand can be reduced It provides a basis for integrating energy efficiency and solar approaches in ways that will allow building owners and designers to balance the need to minimize initial costs operating costs and life cycle costs with need to maintain reliable building operations and enhance environmental quality both inside and outside the building Chapters trace the development of building energy systems and analyze the demand side of solar applications as a means for determining what portion of a building s energy requirements can potentially be met by solar energy Following the introduction the book provides an overview of energy use patterns in the aggregate U S building population Chapter 3 surveys work on the energy flows in an individual building and shows how these flows interact to influence overall energy use Chapter 4 presents the analytical methods techniques and tools developed to calculate and analyze energy use in buildings while chapter 5 provides an extensive survey of the energy conservation and management strategies developed in the post energy crisis period The approach taken is a commonsensical one starting with the proposition that the purpose of buildings is to house human activities and that conservation measures that negatively affect such activities are based on false economies The goal is to determine rational strategies for the design of new buildings and the retrofit of existing buildings to bring them up to modern standards of energy use The energy flows examined are both large scale heating systems and small scale choices among appliances Solar Heat Technologies Fundamentals and Applications Volume 4 Research Paper FPL-RP ,1986 **Index and Directory of U.S. Industry Standards** ,1987

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