

**PROBLEM 3-5**

**Statement:** Design a fourbar mechanism to give the three positions of coupler motion with no-quick return shown in Figure P3-2. (See also Example 3-5.) Ignore the points  $O_2$  and  $O_4$  shown. Build a cardboard model and determine the toggle positions and the minimum transmission angle. Add a driver dyad.

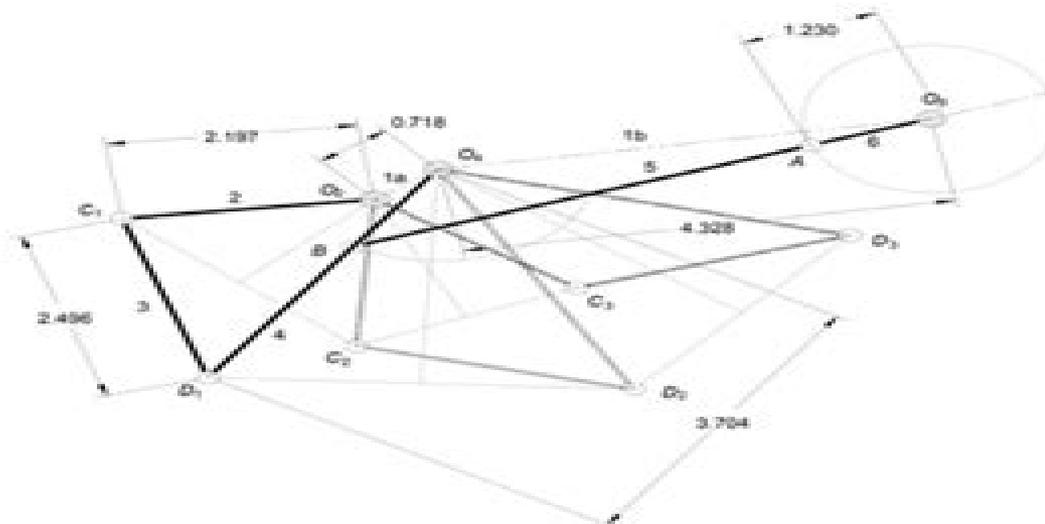
**Solution:** See Figure P3-2 and Mathcad file P0305.

**Design choices:**

$$\text{Length of link 3: } L_3 := 4.250 \qquad \text{Length of link 4b: } L_{4b} := 1.375$$

1. Draw link  $CD$  in its three design positions  $C_1D_1$ ,  $C_2D_2$ ,  $C_3D_3$  in the plane as shown.
2. Draw construction lines from point  $C_1$  to  $C_2$  and from point  $C_2$  to  $C_3$ .
3. Bisect line  $C_1C_2$  and line  $C_2C_3$  and extend their perpendicular bisectors until they intersect. Label their intersection  $O_2$ .
4. Repeat steps 2 and 3 for lines  $D_1D_2$  and  $D_2D_3$ . Label the intersection  $O_4$ .
5. Connect  $O_2$  with  $C_1$  and call it link 2. Connect  $O_4$  with  $D_1$  and call it link 4.
6. Line  $C_1D_1$  is link 3. Line  $O_2O_4$  is link 1 (ground link for the fourbar). The fourbar is now defined as  $O_2CDO_4$  and has link lengths of

Ground link 1a	$L_{1a} := 0.718$	Link 2	$L_2 := 2.197$
Link 3	$L_3 := 2.496$	Link 4	$L_4 := 3.704$



7. Check the Grashof condition. Note that any Grashof condition is potentially acceptable in this case.

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