

Fluid Machinery & Steam Generator

Francis turbine

- Area of flow $A_{f1} = K \pi d_1 b_1$
- Discharge through the runner $Q = A_{f1} V_{f1}$
- Hydraulic power $[HP = \rho g Q H]_{\text{hy}}$
- Runner power when v_{w1} is in direction of u_1
 $R.P = \frac{\rho g Q}{\rho} [v_{w1} u_1 - v_{w2} u_2]$
- Runner power when v_{w1} is in opposite direction of u_2
 $R.P = \frac{\rho g Q}{\rho} [v_{w1} u_1 + v_{w2} u_2]$
- Hydraulic efficiency $\eta_h = \frac{R.P}{HP} = \frac{v_{w1} u_1}{gH}$ L-R-L-R
- Mechanical efficiency $\eta_{mech} = \frac{S.P}{R.P}$ M-S-P
- Overall efficiency $\eta_o = \eta_h \times \eta_{mech}$ O-S-L-R
- Volumetric efficiency $\eta_{vol} = \frac{Q}{Q + \Delta Q}$
 where ΔQ = leakage loss
 Q = discharge entering the turbine
- Degree of reaction (R) : $R = 1 - \frac{v_2^2 - v_1^2}{2 v_{w1} u_1}$
- Efficiency of draft tube $\eta_d = \frac{\frac{v_1^2 - v_2^2}{2g}}{\frac{v_1^2}{2g}}$ η_d
- Pelton wheel**
- Power available at inlet of vanes = $\frac{\rho Q v^2}{2}$
- Runner power $R.P = \frac{\rho g Q}{\rho} [v_{w1} - v_{w2}] u$
- $(\eta_h)_{max} = \frac{1 - k \cos \alpha}{2}$
- Blade efficiency $\eta_{b, max} = \frac{v_1^2 - v_2^2}{v_1^2}$
- Specific speed of Turbine $N_s = \frac{N \sqrt{P}}{H^{5/4}}$

Model relationships for turbine / Cp

Capacity or flow coeff. $\left(\frac{Q}{ND^3} \right)_m = \left(\frac{Q}{ND^3} \right)_p$ velocity coeff. $\left(\frac{gH}{N^2 D^5} \right)_m = \left(\frac{gH}{N^2 D^5} \right)_p$

Power coeff. $\left(\frac{P}{N^3 D^5} \right)_m = \left(\frac{P}{N^3 D^5} \right)_p$

Brayton cycle or Joule cycle:- [G.M.P.P]

$\eta = 1 - \frac{1}{(r_p)^{\gamma}}$ $T_2 = T_4 = \sqrt{T_{max} \cdot T_{min}}$

$(r_p)_{optimum} = \left(\frac{T_{max}}{T_{min}} \right)^{\frac{1}{\gamma-1}}$

$(r_p)_{optimum} = \sqrt{(r_p)_{max}}$ $(r_p)_{max} = \left(\frac{T_{max}}{T_{min}} \right)^{\frac{1}{\gamma-1}}$

Back work Ratio = $\frac{W_c}{W_T}$

Work Ratio = $\frac{W_{net}}{W_T} = \frac{W_T - W_c}{W_T}$ $1 - \frac{W_c}{W_T}$

Rankine vapour power cycle (or) steam power cycle

Specific steam consumption $SSC = \frac{3600}{W_{net}} \frac{kg}{kwhr}$

$\eta_{Rankine} = 1 - \frac{Q_{out}}{Q_{in}} = 1 - \frac{T_c}{T_h} = \frac{W_{net}}{Q_c} = \frac{Q_2 - Q_1}{Q_c} = 1 - \frac{Q_1}{Q_c}$

For regeneration in Rankine cycle

$W_1 = (1-x)(h_1 - h_2) + (1-y)(h_1 - h_2)$

$W_2 = (1-x)(h_1 - h_2) + (1-y)(h_1 - h_2)$

$W_{net} = W_1 - W_2$

$Q_2 = (1-x)(h_2 - h_3)$

Applying conservation of energy

Mechanical Engineering Formula

Navy Feroz



Mechanical Engineering Formula:

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Engineering News and American Contract Journal ,1887 **Journal of the Association of Engineering Societies ...** ,1901 **Engineering News** ,1903 **Methods and Tools for Effective Knowledge Life-Cycle-Management** Alain Bernard,Serge Tichkiewitch,2008-04-01 Knowledge Management is a wide critical and strategic issue for all the com nies from the SMEs to the most complex organizations The key of competiti ness is knowledge because of the necessity of reactivity flexibility agility and innovation capacities Knowledge is difficult to measure itself but what is visible this is the way of improving products technologies and enterprise organizations During the last four

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Mechanical Engineering American Society of Mechanical Engineers, 1919 History of the American society of mechanical engineers Preliminary report of the committee on Society history issued from time to time beginning with v 30 Feb 1908

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