

Principal Stress & Maximal Shear Stress

2-Dimensional

$$\sigma_{1,2} = \frac{\sigma_x + \sigma_y}{2} \pm \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2}$$

$$\tau_{max} = \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2}$$

$$\sigma_{x'} = \frac{\sigma_x + \sigma_y}{2} + \left(\frac{\sigma_x - \sigma_y}{2}\right) \cos 2\theta + \tau_{xy} \sin 2\theta$$

$$\sigma_{y'} = \frac{\sigma_x + \sigma_y}{2} - \left(\frac{\sigma_x - \sigma_y}{2}\right) \cos 2\theta + \tau_{xy} \sin 2\theta$$

$$\tau_{x'y'} = -\left(\frac{\sigma_x - \sigma_y}{2}\right) \sin 2\theta + \tau_{xy} \cos 2\theta$$

$$\theta = \frac{1}{2} \tan^{-1} \left(\frac{2\tau_{xy}}{\sigma_x - \sigma_y} \right)$$

$$I_{xx} = \int y^2 dA, I_{yy} = \int x^2 dA, I_{xy} = \int xy dA, I_{x'x'} = I_{xx} \cos^4 \theta - 2I_{xy} \cos^2 \theta \sin 2\theta + I_{yy} \sin^4 \theta$$

3-Dimensional

Determinants Equations (Stress Invariants)

$$I_1 = \sigma_x + \sigma_y + \sigma_z$$

$$I_2 = \sigma_x \sigma_y + \sigma_y \sigma_z + \sigma_z \sigma_x - \tau_{xy}^2 - \tau_{yz}^2 - \tau_{zx}^2$$

$$I_3 = \sigma_x \sigma_y \sigma_z + 2\tau_{xy} \tau_{yz} \tau_{zx} - \sigma_x \tau_{xy}^2 - \sigma_y \tau_{yz}^2 - \sigma_z \tau_{zx}^2 - 3\tau_{xy} \tau_{yz} \tau_{zx}$$

$$I_1^3 - 3I_1 I_2 + 3I_3 = 0 \Rightarrow \sigma_1, \sigma_2, \sigma_3$$

Principle stress of 1, 2, 3, where

$$\sigma_{max} = \frac{I_1 + \sqrt{I_1^2 - 3I_2}}{3}$$

See text for principal direction

$$\sigma_{min} = \frac{I_1 - \sqrt{I_1^2 - 3I_2}}{3}$$

$$\sigma_{int} = \frac{I_1 - \sqrt{I_1^2 - 3I_2}}{3}$$

Centroidal Bending

$$\sigma_{max} = \frac{M y}{I}$$

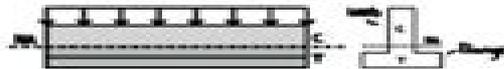
C is distance from NB to point of interest; y is taken for counter-clockwise

I is NB with the MCT neutral axis

bending stress will be MCT (negative) side

MCA is neutral (deformed) stress distribution

tensile bending → positive, compression bending → negative

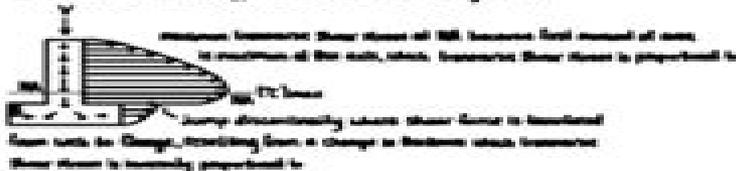


Transverse Shear

$$\tau = \frac{VQ}{It}$$

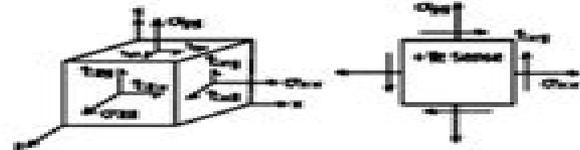
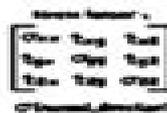
Q → Area, for rectangles along axis but I & Q are with respect to

I & Q will be MCT axis opposite to V (other axis (horizontal))



Torsion

$$\tau = \frac{T\rho}{J}$$



Case Sectional Properties

$$Centroid: \bar{y} = \frac{\sum y_i A_i}{\sum A_i}$$

Second Moment of Area

$$I = \sum I_{xxi} + \sum A_i d_i^2$$

$$d_i = (y_i - \bar{y})$$

If I_{xy}, have that in face perspective of y₁-axis

If I_{xy}, " " z₁-axis

Area Moment of Inertia

$$J = \sum (I_{xxi} + I_{yyi})$$

I_{xx} = I_{yy} if bar is used as a shaft; hollow one: I_{xx} = I_{yy}

First Moment of Area

moment of area above or below point of interest

$$Q = A y$$

where A is the remaining case sectional area above or below that point

and y is the distance between the centroid of that area to the centroid of the entire case section

2-Dimensional Plane Stress & Strain

Stress

(1) — one dimension compared to other two dimensions is very small

(2) — like a plate or a shell, stress with large constant

(3) — total has to be in the plane of the plate, etc., same way it have max.

Strain

(1) — one dimension compared to other two dimensions is very large

(2) — symmetric loading condition

Stress-Strain

A: point of interest

$$\begin{bmatrix} \sigma_x \\ \sigma_y \\ \tau_{xy} \end{bmatrix} = \begin{bmatrix} C_{11} & C_{12} & C_{13} \\ C_{21} & C_{22} & C_{23} \\ C_{31} & C_{32} & C_{33} \end{bmatrix} \begin{bmatrix} \epsilon_x \\ \epsilon_y \\ \gamma_{xy} \end{bmatrix} \rightarrow \begin{bmatrix} E_{11} & E_{12} & E_{13} \\ E_{21} & E_{22} & E_{23} \\ E_{31} & E_{32} & E_{33} \end{bmatrix} \begin{bmatrix} \epsilon_x \\ \epsilon_y \\ \gamma_{xy} \end{bmatrix} \rightarrow \begin{bmatrix} E_{11} & E_{12} & E_{13} \\ E_{21} & E_{22} & E_{23} \\ E_{31} & E_{32} & E_{33} \end{bmatrix}$$

Stress-Strain Relationship

$$E_{xx} = E(\sigma_x/\epsilon_{xx} - \nu\sigma_y/\epsilon_{yy} - \nu\sigma_z/\epsilon_{zz})$$

$$E_{yy} = E(\sigma_y/\epsilon_{yy} - \nu\sigma_x/\epsilon_{xx} - \nu\sigma_z/\epsilon_{zz})$$

$$E_{zz} = E(\sigma_z/\epsilon_{zz} - \nu\sigma_x/\epsilon_{xx} - \nu\sigma_y/\epsilon_{yy})$$

$$G_{xy} = E(\sigma_x/\tau_{xy} + \sigma_y/\tau_{xy}) ; G_{yz} = E(\sigma_y/\tau_{yz} + \sigma_z/\tau_{yz}) ; G_{zx} = E(\sigma_z/\tau_{zx} + \sigma_x/\tau_{zx})$$

$$G = E/(2(1+\nu))$$

Plane Stress Condition

$$E_{xx} = E(\sigma_x/\epsilon_{xx} - \nu\sigma_y/\epsilon_{yy})$$

$$E_{yy} = E(\sigma_y/\epsilon_{yy} - \nu\sigma_x/\epsilon_{xx})$$

$$E_{zz} = -\nu E(\sigma_x/\epsilon_{xx} + \sigma_y/\epsilon_{yy})$$

$$G_{xy} = E/(2(1+\nu))$$

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1. Understanding the eBook Course Notes Structural Mechanics Mechanical
 - The Rise of Digital Reading Course Notes Structural Mechanics Mechanical
 - Advantages of eBooks Over Traditional Books
2. Identifying Course Notes Structural Mechanics Mechanical
 - Exploring Different Genres
 - Considering Fiction vs. Non-Fiction
 - Determining Your Reading Goals
3. Choosing the Right eBook Platform
 - Popular eBook Platforms
 - Features to Look for in an Course Notes Structural Mechanics Mechanical
 - User-Friendly Interface
4. Exploring eBook Recommendations from Course Notes Structural Mechanics Mechanical
 - Personalized Recommendations
 - Course Notes Structural Mechanics Mechanical User Reviews and Ratings
 - Course Notes Structural Mechanics Mechanical and Bestseller Lists
5. Accessing Course Notes Structural Mechanics Mechanical Free and Paid eBooks
 - Course Notes Structural Mechanics Mechanical Public Domain eBooks
 - Course Notes Structural Mechanics Mechanical eBook Subscription Services
 - Course Notes Structural Mechanics Mechanical Budget-Friendly Options
6. Navigating Course Notes Structural Mechanics Mechanical eBook Formats

- ePub, PDF, MOBI, and More
 - Course Notes Structural Mechanics Mechanical Compatibility with Devices
 - Course Notes Structural Mechanics Mechanical Enhanced eBook Features
7. Enhancing Your Reading Experience
 - Adjustable Fonts and Text Sizes of Course Notes Structural Mechanics Mechanical
 - Highlighting and Note-Taking Course Notes Structural Mechanics Mechanical
 - Interactive Elements Course Notes Structural Mechanics Mechanical
 8. Staying Engaged with Course Notes Structural Mechanics Mechanical
 - Joining Online Reading Communities
 - Participating in Virtual Book Clubs
 - Following Authors and Publishers Course Notes Structural Mechanics Mechanical
 9. Balancing eBooks and Physical Books Course Notes Structural Mechanics Mechanical
 - Benefits of a Digital Library
 - Creating a Diverse Reading Collection Course Notes Structural Mechanics Mechanical
 10. Overcoming Reading Challenges
 - Dealing with Digital Eye Strain
 - Minimizing Distractions
 - Managing Screen Time
 11. Cultivating a Reading Routine Course Notes Structural Mechanics Mechanical
 - Setting Reading Goals Course Notes Structural Mechanics Mechanical
 - Carving Out Dedicated Reading Time
 12. Sourcing Reliable Information of Course Notes Structural Mechanics Mechanical
 - Fact-Checking eBook Content of Course Notes Structural Mechanics Mechanical
 - Distinguishing Credible Sources
 13. Promoting Lifelong Learning
 - Utilizing eBooks for Skill Development
 - Exploring Educational eBooks
 14. Embracing eBook Trends
 - Integration of Multimedia Elements
 - Interactive and Gamified eBooks

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